

FORMULAIRE MEC2405

Section 1 : révision MEC1420

$$\sigma_x = \frac{F}{A}$$

$$\sigma_x = -\frac{M_z y}{I_z} = -\frac{M_z}{S_z}$$

$$\tau_{x\theta} = \frac{Tr}{J}$$

Torsion section ouverte

$$\tau_{xi} = \frac{Tt}{J} \quad J = \sum \frac{bt^3}{3}$$

$$\sigma_{x'} = \frac{\sigma_x + \sigma_y}{2} + \frac{\sigma_x - \sigma_y}{2} \cos(2\theta) + \tau_{xy} \sin(2\theta)$$

$$\sigma_{1,2} = \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$

$$\sigma_{x'} + \sigma_{y'} = \sigma_x + \sigma_y = \sigma_1 + \sigma_2$$

$$\sigma_x = \frac{Pr_m}{2t}$$

$$I = \frac{bh^3}{12}$$

$$\tau = \frac{VQ}{Ib}$$

$$\sigma_\theta = \frac{Pr_m}{t}$$

$$I = \frac{\pi r^4}{4}$$

$$J = \frac{\pi r^4}{2}$$

$$I = \pi r^3 t$$

$$J = 2\pi r^3 t$$

Torsion section fermée

$$\tau_{xs} = \frac{T}{2\bar{A}t} \quad J = \frac{4\bar{A}^2}{\int \frac{ds}{t}}$$

$$\tau_{x'y'} = -\frac{\sigma_x - \sigma_y}{2} \sin(2\theta) + \tau_{xy} \cos(2\theta)$$

$$\tan(2\theta_1) = \frac{2\tau_{xy}}{\sigma_x - \sigma_y}$$

Section 2 : flexion gauche

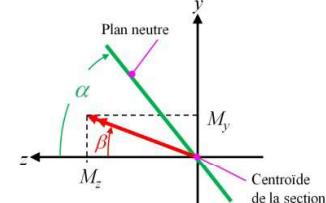
$$\bar{y}_i = \frac{\sum A_i \bar{y}_i}{\sum A_i}$$

$$\tan \beta = \frac{M_y}{M_z}$$

$$\tan \alpha = \frac{M_z I_{yz} + M_y I_z}{M_y I_{yz} + M_z I_y} = \frac{I_{yz} + I_z \tan \beta}{I_{yz} \tan \beta + I_y}$$

$$\sigma_x = -\frac{1}{I_y I_z - I_{yz}^2} \left[(M_y I_{yz} + M_z I_y) y - (M_z I_{yz} + M_y I_z) z \right]$$

$$\sigma_x = -\left[\frac{M_y}{I_{yz}^*} + \frac{M_z}{I_z^*} \right] y + \left[\frac{M_z}{I_{yz}^*} + \frac{M_y}{I_y^*} \right] z$$



$$I_y^* = \frac{I_y I_z - I_{yz}^2}{I_z} \quad ; \quad I_z^* = \frac{I_y I_z - I_{yz}^2}{I_y} \quad ; \quad I_{yz}^* = \frac{I_y I_z - I_{yz}^2}{I_{yz}}$$

$$I_{yz} = \sum (I_{y_i z_i} + \bar{y}_i \bar{z}_i A)$$

$$I_{y'} = \frac{I_y + I_z}{2} + \frac{I_y - I_z}{2} \cos 2\theta - I_{yz} \sin 2\theta$$

$$I_{z'} = \frac{I_y + I_z}{2} - \frac{I_y - I_z}{2} \cos 2\theta + I_{yz} \sin 2\theta$$

$$I = Ar^2$$

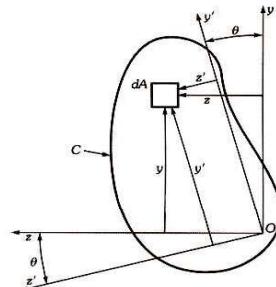


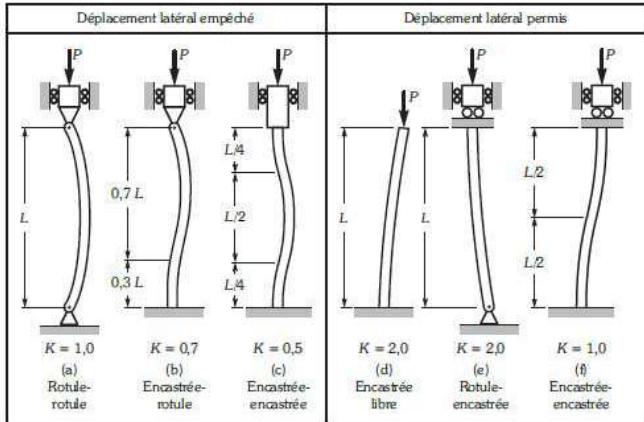
Figure A.4 Réaction du système d'axes.
Le système d'axes y' et z' fait avec le système d'axes y et z un angle θ .

Section 3 : Instabilité et flambement

$$P_{\text{cr}} = \frac{\pi^2 EI}{(KL)^2}$$

$$\sigma_{\text{cr}} = \frac{P_{\text{cr}}}{A} = \frac{\pi^2 E}{\left(\frac{KL}{r}\right)^2}$$

$$r = \sqrt{\frac{I}{A}}$$



Code ACNOR

Flexion 2 plans

$$C_r = \phi A S_y \left(1 + \lambda^{2n}\right)^{-1} \quad \lambda = \frac{KL}{r} \sqrt{\frac{S_y}{\pi^2 E}}$$

$$F_{\text{amp}} = \frac{1}{1 - \frac{C}{P_{\text{cr}}}}$$

Flexion 1 plan

$$\frac{C}{C_r} + \frac{F_{\text{amp}} M_z}{M_{rz}} + \frac{F_{\text{amp}} M_y}{M_{ry}} \leq 1$$

$$\frac{C}{C_i} + \frac{F_{\text{amp}/i} M_i}{M_{ri}} \leq 1,0$$

Section 4 : Énergie de déformation

Énergie de déformation

$$U = \sum \frac{P_i^2 L}{2AE} + \sum \frac{T_i^2 L}{2GJ} + \sum \int \frac{M_i^2 dx}{2EI} + \sum \int \frac{V_i^2 dx}{2A_c G}$$

Théorème de Castigliano

$$\frac{dU}{dF} = \sum P_i \frac{\partial P_i}{\partial F} L + \sum T_i \frac{\partial T_i}{\partial F} L + \sum \int M_i \frac{\partial M_i}{\partial F} dx + \sum \int V_i \frac{\partial V_i}{\partial F} dx$$

Théorème de Maxwell-Betti

$$\sum_{i=1}^n (\bar{P}_i)_I \cdot (\bar{\delta}_i)_{II} = \sum_{j=1}^m (\bar{P}_j)_{II} \cdot (\bar{\delta}_j)_I$$